



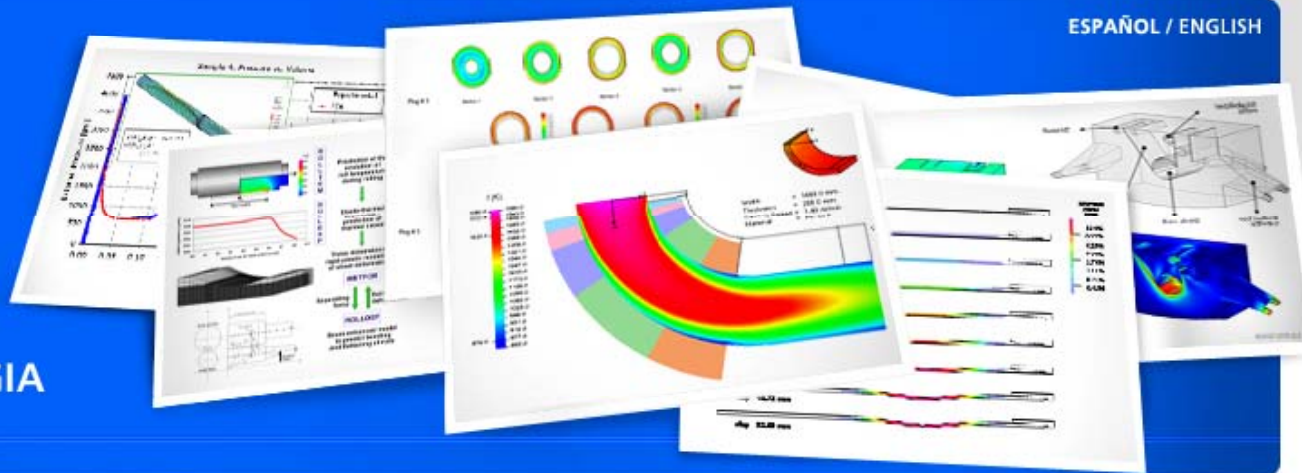
# SIM&TEC

Simulación y Tecnología

*Simulation and Technology*

ESPAÑOL / ENGLISH

DE LA CIENCIA  
A LA TECNOLOGIA



# Advanced Topics in Computational Solid Mechanics. Industrial Applications

## Section 9 : Tracking Nonlinear Equilibrium paths: The Riks Method

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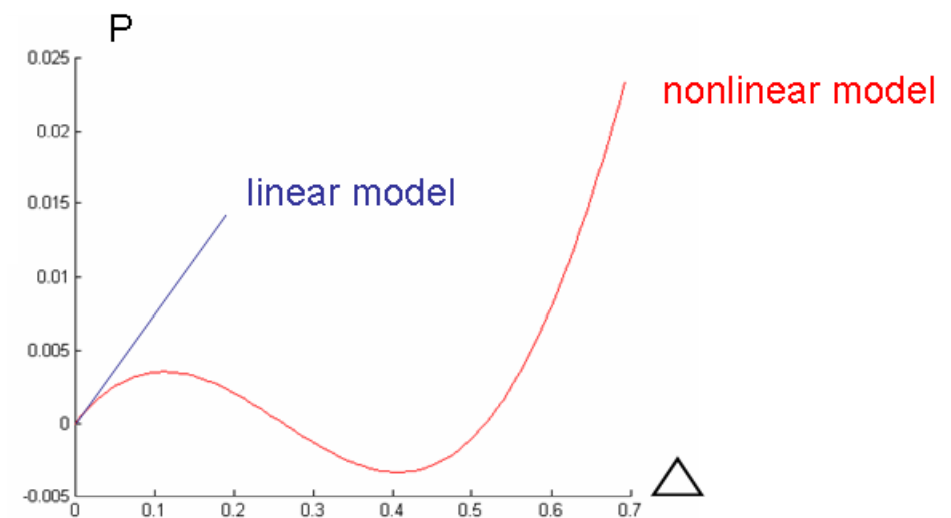
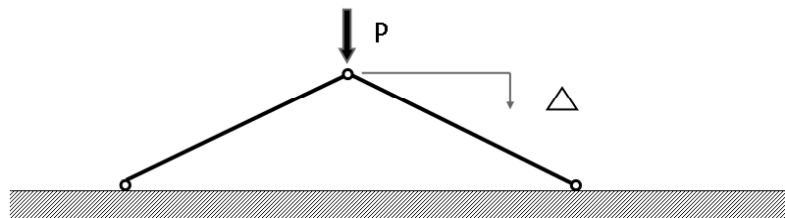
# Load Control Solution

$${}^{t+\Delta t}\mathbf{R} - {}^{t+\Delta t}\mathbf{F} = \mathbf{0}$$

$${}^{t+\Delta t}\mathbf{K}^{(i-1)}\Delta\mathbf{U}^{(i)} = {}^{t+\Delta t}\mathbf{R} - {}^{t+\Delta t}\mathbf{F}^{(i-1)}$$

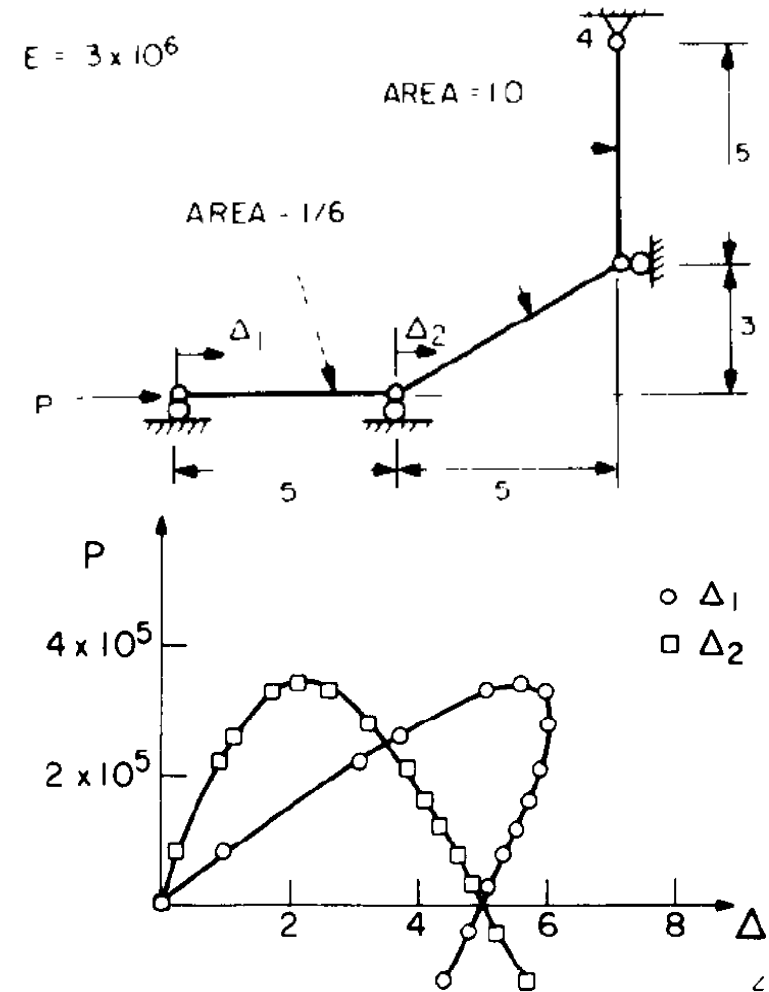
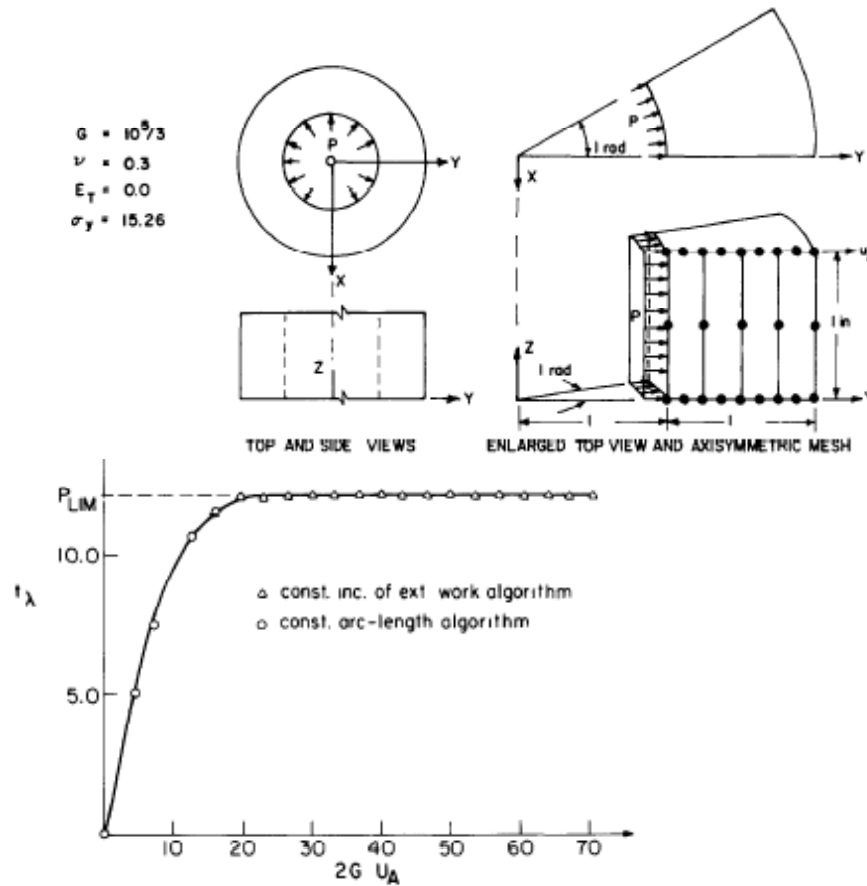
$${}^{t+\Delta t}\mathbf{U}^{(i)} = {}^{t+\Delta t}\mathbf{U}^{(i-1)} + \Delta\mathbf{U}^{(i)}$$

Fails in:



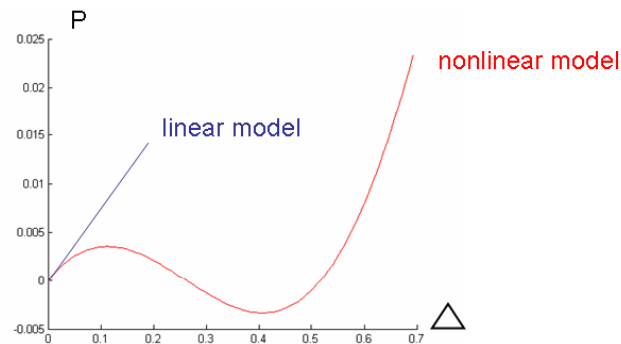
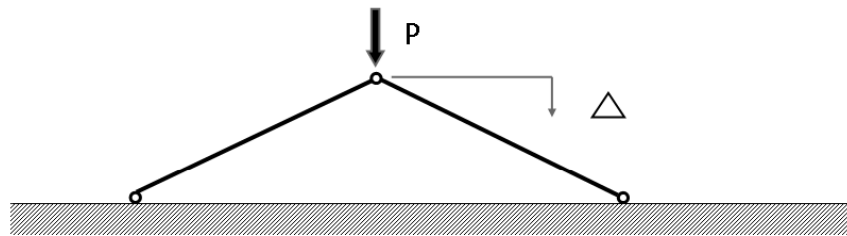
# Load Control Failures

Elasto-perfectly plastic material.  
 Plane Strain conditions. Von-mises yield condition

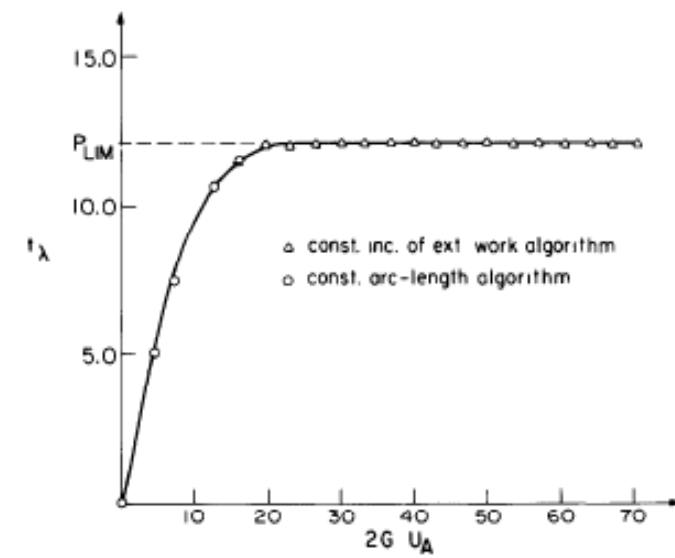
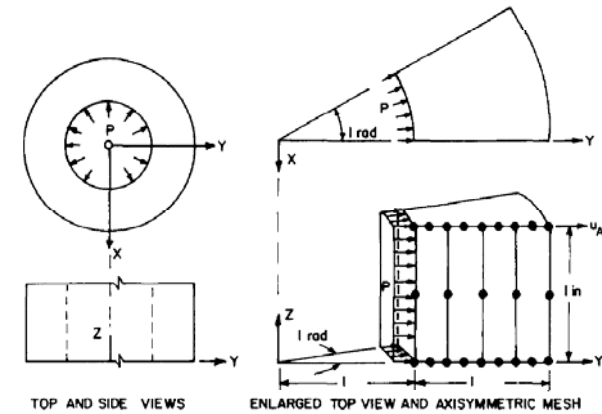


# Displacement Control Solution

o.k. for:

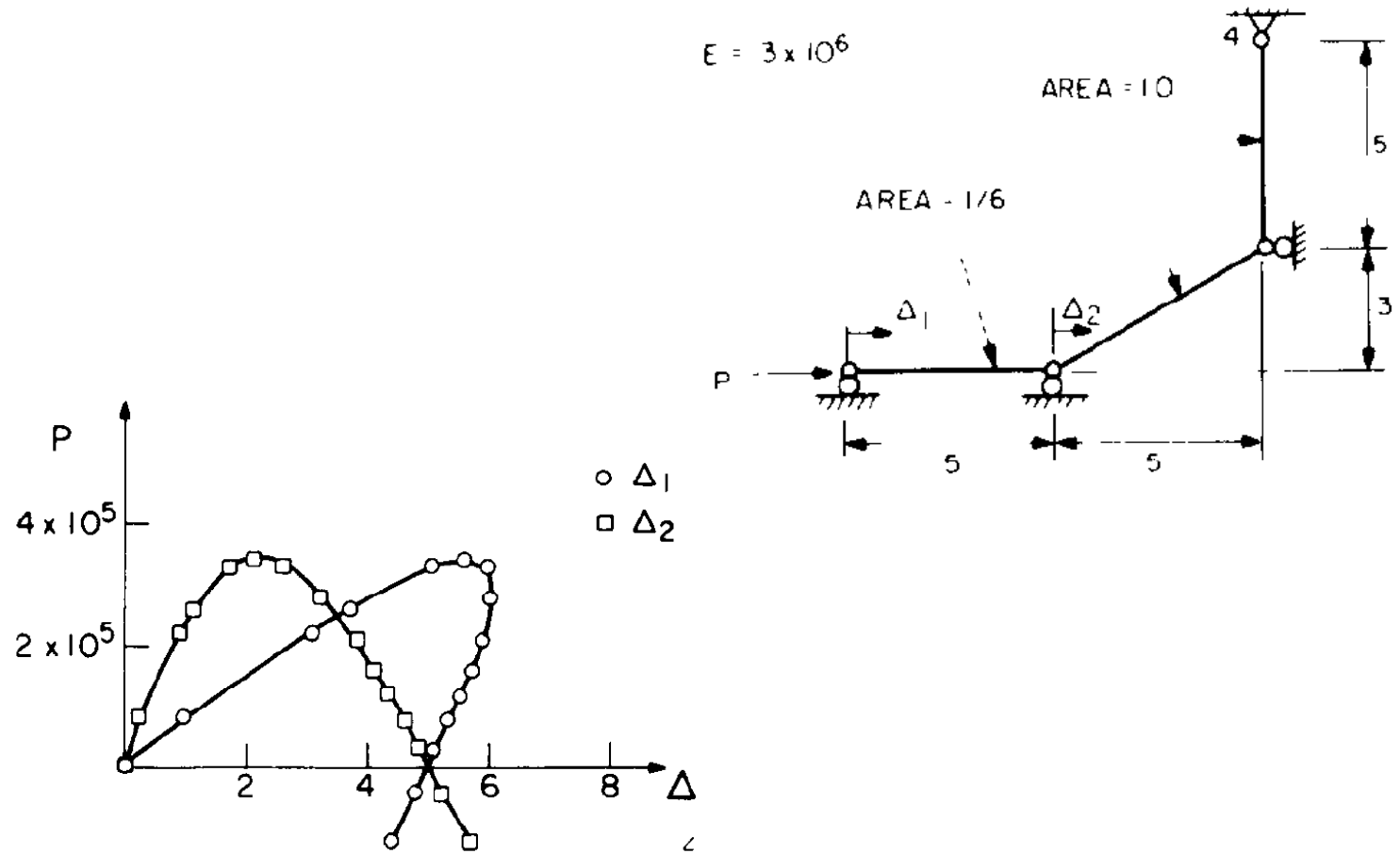


$G = 10^{9.3}$   
 $\nu = 0.3$   
 $E_T = 0.0$   
 $\sigma_y = 15.26$



# Displacement Control Solution

Fails for:



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# The Riks Method: Iterate in the Load - Displacement Space

$${}^t\mathbf{K} \Delta\mathbf{U}^{(i)} = {}^{t+\Delta t}\lambda \mathbf{R} - {}^{t+\Delta t}\mathbf{F}^{(i-1)}$$

$${}^t\mathbf{K} \Delta\mathbf{U}^{(i)} = ({}^{t+\Delta t}\lambda^{(i-1)} + \Delta\lambda^{(i)})\mathbf{R} - {}^{t+\Delta t}\mathbf{F}^{(i-1)}$$

and an additional equation is employed to constrain the length of the load step

$$f(\Delta\lambda^{(i)}, \Delta\mathbf{U}^{(i)}) = 0$$

## The Riks Method: Arc Length

We use the spherical constant arc-length in the response regions far from the critical points

$$\{({}^{t+\Delta t}\lambda^{(i-1)} - {}^t\lambda) + \Delta\lambda^{(i)}\}^2 + \mathbf{U}^{(i)T} \mathbf{U}^{(i)} = \Delta l^2 \quad (7)$$

$$\mathbf{U}^{(i)} = {}^{t+\Delta t}\mathbf{U}^{(i)} - {}^t\mathbf{U}$$

where  $\Delta l$  is the arc-length.



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## The Riks Method: Constant Increment of External Work

The scheme of constant increment of external work is used near the critical points. In this case eqn (6) is for the first iteration,

$$({}^t\lambda + \frac{1}{2} \Delta\lambda^{(1)})\mathbf{R}^T \Delta\mathbf{U}^{(1)} = W \quad (8a)$$

and for the next iterations,

$$({}^{t+\Delta t}\lambda^{(i-1)} + \frac{1}{2} \Delta\lambda^{(i)})\mathbf{R}^T \Delta\mathbf{U}^{(i)} = 0 \quad i = 2, 3, \dots \quad (8b)$$

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# The Riks Method: Practical Implementation

- (1) The user inputs the reference load distribution, which corresponds to the vector  $R$ . This load distribution is varied proportionally during the analysis and can be due to distributed and concentrated loads.
- (2) The user specifies the displacement at a node corresponding to the first load level (i.e. corresponding to  $\Delta^t \lambda$ ). We denote this displacement as  $\Delta^t U_k^*$ . Here it is deemed that to start the incremental solution it is easier to specify the displacement at a node, that the user selects, than the intensity ( $\Delta^t \lambda$ ) of the loads.
- (3) The displacements corresponding to time  $\Delta t$  determined by the specified displacement  $\Delta^t U_k^*$  also limit the size of any subsequent load change per step, because the user specifies a constant  $\alpha$  and the algorithm assures that

$$\|\mathbf{U}\| \leq \alpha \|\Delta^t \mathbf{U}\| \quad (9)$$

where  $\mathbf{U}$  is the displacement increment in any load step and  $\Delta^t \mathbf{U}$  are the displacements corresponding to time  $\Delta t$ .

# The Riks Method: Practical Implementation

First step

$${}^0\mathbf{K} \Delta\mathbf{U}^{(i)} = (\Delta^t\lambda^{(i-1)} + \Delta\lambda^{(i)})\mathbf{R} - \Delta^t\mathbf{F}^{(i-1)}$$

$$\Delta^t\lambda^{(0)} = 0; \Delta^t\mathbf{F}^{(0)} = \mathbf{0}$$

$${}^0\mathbf{K} \Delta\mathbf{U}^{(1)} = \mathbf{R}$$

$$\Delta^t\lambda^{(1)} = \frac{\Delta^t U_k^*}{\Delta U_k^{(1)}}; \Delta^t\mathbf{U}^{(1)} = \Delta^t\lambda^{(1)} \Delta\mathbf{U}^{(1)}$$

$i=2,3,\dots$

$${}^0\mathbf{K} \Delta\bar{\mathbf{U}}^{(i)} = \Delta^t\lambda^{(i-1)} \mathbf{R} - \Delta^t\mathbf{F}^{(i-1)} \quad \Delta\bar{\mathbf{U}}^{(i)} = \Delta\lambda^{(i)} \Delta\mathbf{U}^{(1)}$$

$${}^0\mathbf{K} \Delta\bar{\mathbf{U}}^{(i)} = \Delta\lambda^{(i)} \mathbf{R}$$

$$\Delta\lambda^{(i)} = -\frac{\Delta\bar{U}_k^{(i)}}{\Delta U_k^{(1)}}$$

$$\Delta^t\lambda^{(i)} = \Delta^t\lambda^{(i-1)} + \Delta\lambda^{(i)}$$

$$\Delta^t\mathbf{U}^{(i)} = \Delta^t\mathbf{U}^{(i-1)} + \Delta\mathbf{U}^{(i)}$$

$$\Delta\mathbf{U}^{(i)} = \Delta\bar{\mathbf{U}}^{(i)} + \Delta\lambda^{(i)} \Delta\mathbf{U}^{(1)}$$

# The Riks Method: Practical Implementation

$2\Delta t$ ;  $3\Delta t$ ; ... (arc length)

$$\Delta l = \beta \sqrt{\mathbf{U}^T \mathbf{U} + \lambda^2}$$

$$\mathbf{U} = {}^t\mathbf{U} - {}^{t-\Delta t}\mathbf{U}; \lambda = {}^t\lambda - {}^{t-\Delta t}\lambda$$

$${}^t\mathbf{K} \Delta \bar{\mathbf{U}}^{(1)} = {}^t\lambda \mathbf{R} - {}^t\mathbf{F}$$

$$\Delta \mathbf{U}^{(1)} = \Delta \bar{\mathbf{U}}^{(1)} + \Delta \lambda^{(1)} \Delta \bar{\bar{\mathbf{U}}}^{(1)}$$

$${}^t\mathbf{K} \Delta \bar{\bar{\mathbf{U}}}^{(1)} = \mathbf{R}$$

$${}^{t+\Delta t}\mathbf{U}^{(1)} = {}^t\mathbf{U} + \Delta \mathbf{U}^{(1)}; {}^{t+\Delta t}\lambda^{(1)} = {}^t\lambda + \Delta \lambda^{(1)}$$

$$\Delta \mathbf{U}^{(1)T} \Delta \mathbf{U}^{(1)} + (\Delta \lambda^{(1)})^2 = \Delta l^2, \quad \text{select one of the roots for } \Delta \lambda$$

$i=2, 3, \dots$

$${}^t\mathbf{K} \Delta \bar{\mathbf{U}}^{(i)} = {}^{t+\Delta t}\lambda^{(i-1)} \mathbf{R} - {}^{t+\Delta t}\mathbf{F}^{(i-1)}$$

$${}^{t+\Delta t}\mathbf{U}^{(i)} = {}^{t+\Delta t}\mathbf{U}^{(i-1)} + \Delta \bar{\mathbf{U}}^{(i)} + \Delta \lambda^{(i)} \Delta \bar{\bar{\mathbf{U}}}^{(1)} \quad {}^{t+\Delta t}\lambda^{(i)} = {}^{t+\Delta t}\lambda^{(i-1)} + \Delta \lambda^{(i)}$$

# The Riks Method: Practical Implementation

$$\Delta l|_{\text{new}} = \Delta l|_{\text{old}} \sqrt{\frac{N_1}{N_2} \frac{\|\mathbf{U}\|_{\text{allowable}}}{\|\mathbf{U}\|_{\text{actual}}}}$$

# The Riks Method: Practical Implementation

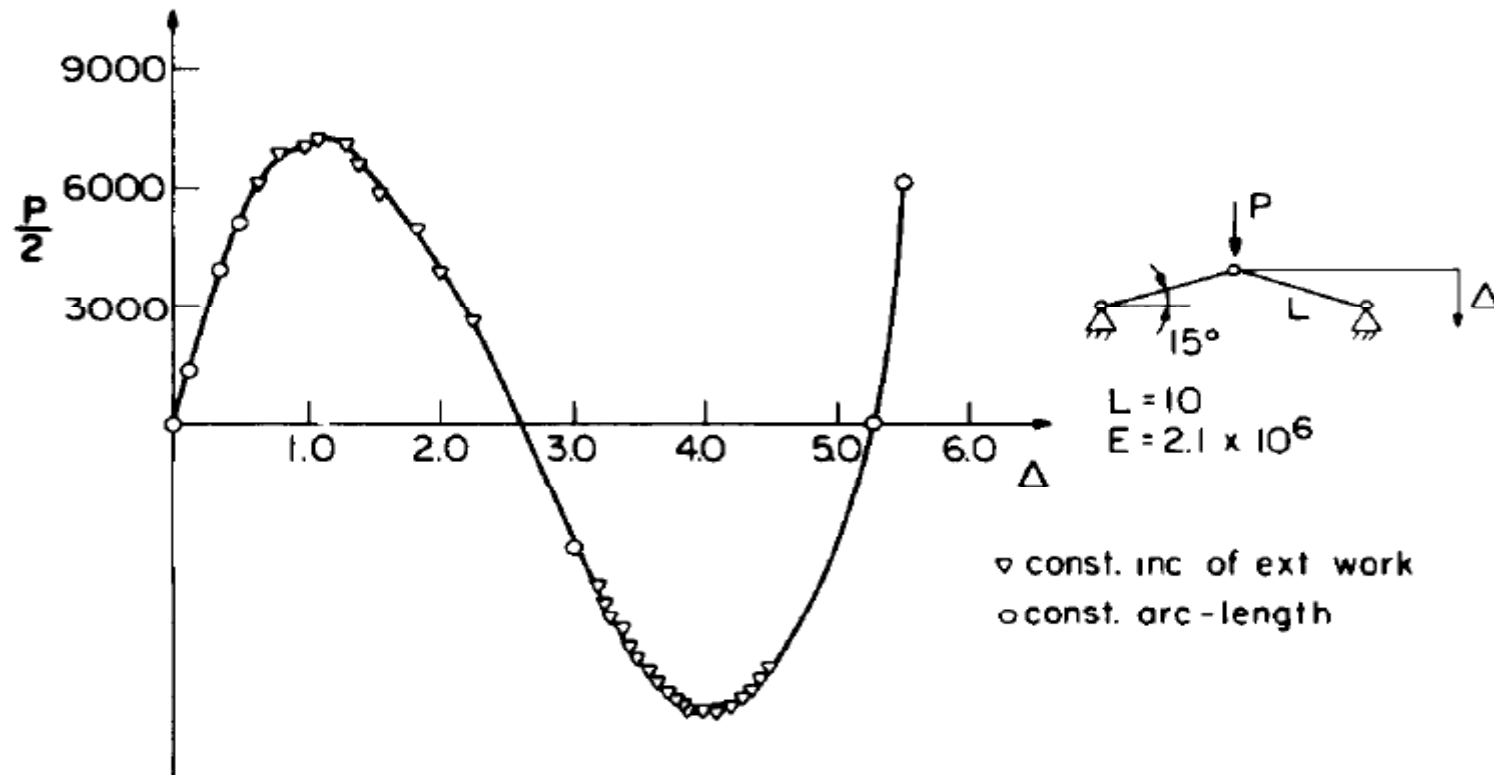
$2\Delta t$ ;  $3\Delta t$ ; ... (constant increment of external work)

$${}^{t+\Delta t}W = \beta {}^tW$$

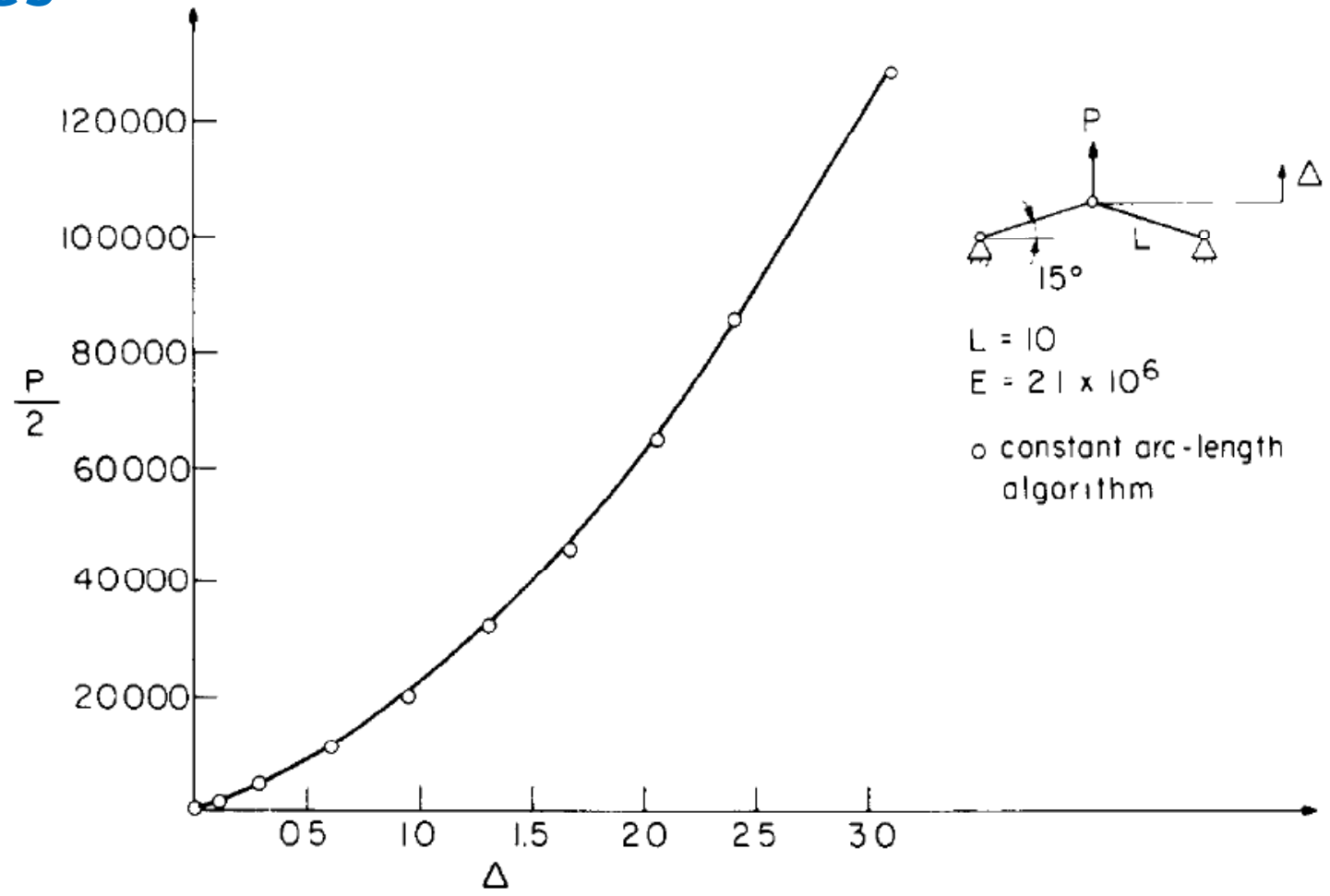
$$({}^t\lambda + \frac{1}{2} \Delta\lambda^{(1)})\mathbf{R}^T \Delta\mathbf{U}^{(1)} = W$$

$$({}^{t+\Delta t}\lambda^{(i-1)} + \frac{1}{2} \Delta\lambda^{(i)})\mathbf{R}^T \Delta\mathbf{U}^{(i)} = 0$$

# Examples



# Examples



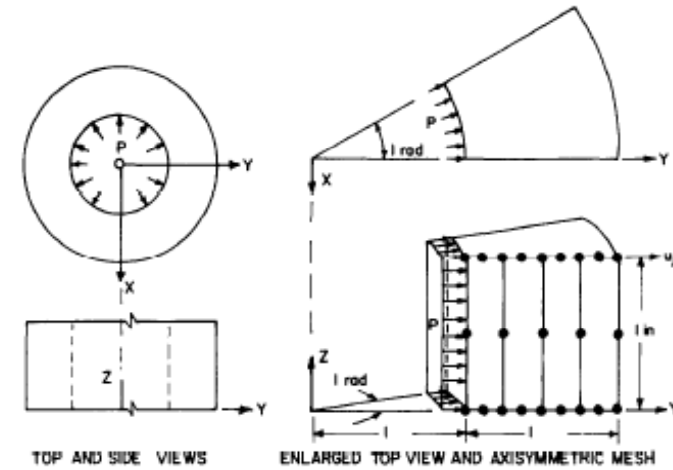
b) Stiffening problem



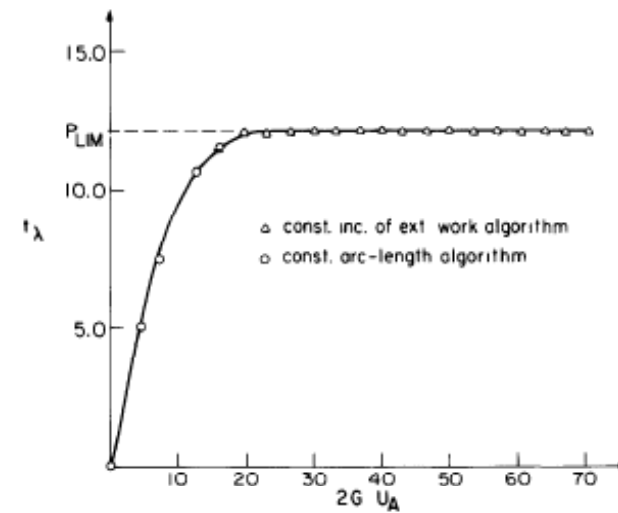
# Examples

## Thick-walled elastic perfectly plastic cylinder

$G = 10^{5/3}$   
 $\nu = 0.3$   
 $E_T = 0.0$   
 $\sigma_y = 15.26$



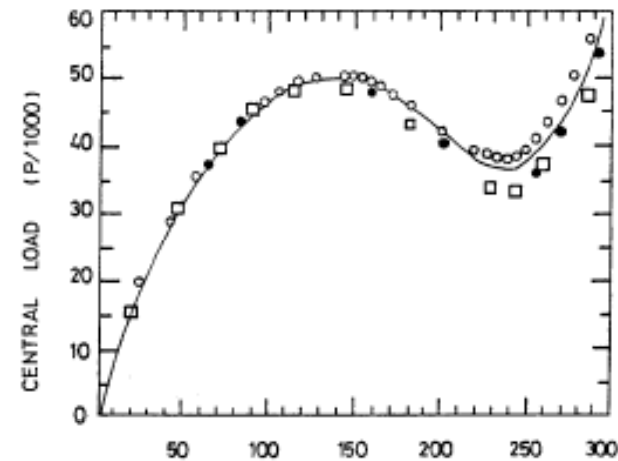
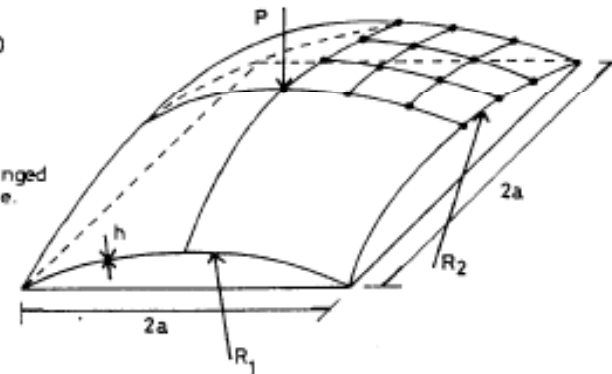
TOP AND SIDE VIEWS      ENLARGED TOP VIEW AND AXISYMMETRIC MESH  
 ELASTIC-PERFECTLY PLASTIC MATERIAL  
 PLANE STRAIN CONDITIONS  
 VON-MISES YIELD CONDITION



# Examples

## Nonlinear spherical shell (MITC4)

$R_1 = R_2 = 2540$   
 $a = 784.90$   
 $h = 99.45$   
 $E = 68.95$   
 $\nu = 0.3$   
 All edges are hinged  
 and immovable.

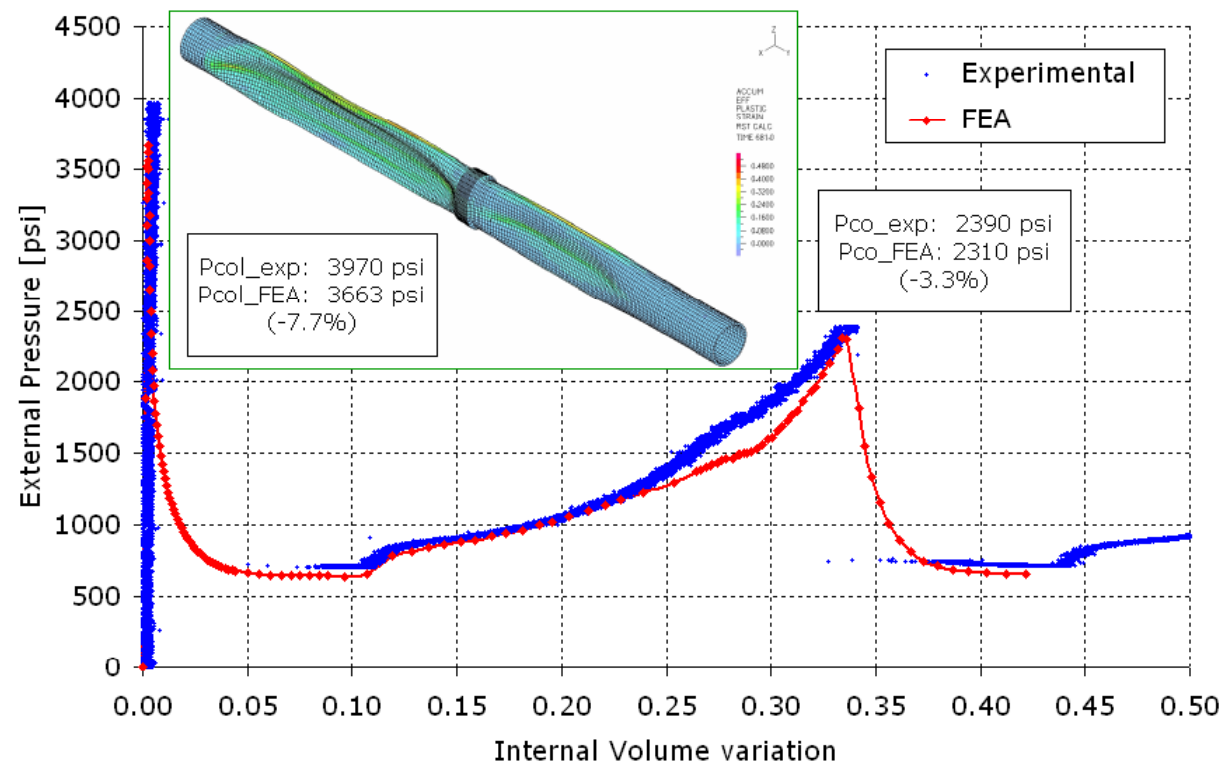


- Horrigoe
- Leicester
- Nine 4-node elements.
- One 16-node element (Std.A-I-Z) Int. 4x4x2.

# Examples

## Collapse

Sample 4: Pressure vs. Volume



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# Iterative Methods

We have plenty of freedom to select the matrix

because we just have to get,

$$[{}^tR] - [{}^tF]_{(k-1)} < RTOL$$

independently of the iteration path.

Iterative methods: full Newton, modified Newton, BFGS.  
Combine with line searches.

# Iteration Tolerances

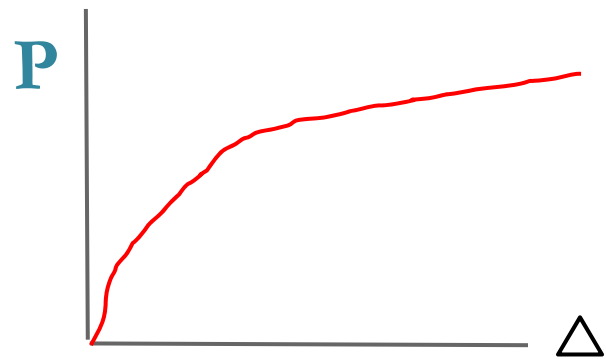
A fundamental decision:

- ▶ If too restrictive we may not get a solution
- ▶ If too ample we may get a very bad solution

$$\begin{aligned} [{}^tR] - [{}^tF]_{(k-1)} &< RTOL \\ \left\| [\Delta \underline{U}]_{(k)} \right\| &< UTOL \\ \left( [\Delta \underline{U}]_{(k)} \right)^T \left( [{}^tR] - [{}^tF]_{(k-1)} \right) &< ETOL \end{aligned}$$

# Iteration Tolerances

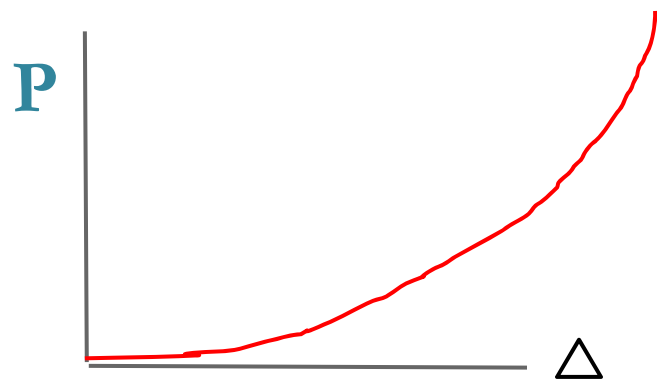
Examples:



Softening problem

$$[{}^tR] - [{}^tF]_{(k-1)} < RTOL$$

Alone is not enough



Stiffening problem

$$\|[\Delta U]_{(k)}\| < UTOL$$

Alone is not enough